

Control Systems And Their Components (EE391)

Lec. 10: Closed loop SS Control (Reference Inputs and State Estimators)

Thu. April 28th, 2016

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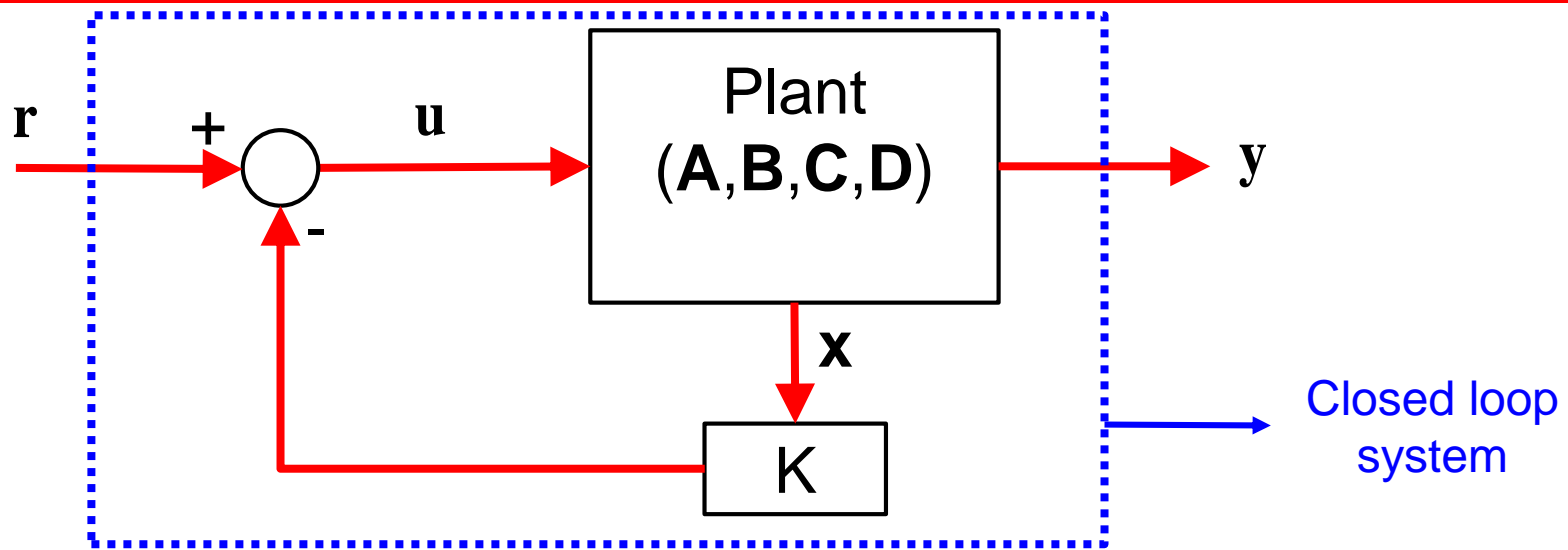
Lecture Outline

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- Adding reference inputs to the design of state feedback controllers
- Pre-scaling reference inputs to achieve zero steady state error
- Introduction to state estimators/observers

Full state feedback (from last lecture)

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$$\dot{\mathbf{x}}(t) = \mathbf{A}\mathbf{x}(t) + \mathbf{B}\mathbf{u}(t)$$

Assume full state feedback of the form

$$\mathbf{u}(t) = \mathbf{r}(t) - \mathbf{K}\mathbf{x}(t)$$

where \mathbf{r} is a reference input and $\mathbf{K} \in \mathbf{R}^{1 \times n}$ (assume a single input for simplicity)

Full state feedback with reference input

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$$\dot{\mathbf{x}}(t) = \underbrace{(\mathbf{A} - \mathbf{BK})}_{\mathbf{A}_{cl}} \mathbf{x}(t) + \mathbf{B}\mathbf{r}(t)$$

Now we will not assume $\mathbf{r} = 0$ as we did in regulator case, and assume we have a certain reference input $\mathbf{r} \neq 0$ that we would like the output \mathbf{y} to track (what does this mean?)

$$\mathbf{y}(t) = \mathbf{C}\mathbf{x}(t), \quad \text{assume } \mathbf{D} = 0$$

A sufficient condition on the output \mathbf{y} for tracking \mathbf{r} is

$$\lim_{t \rightarrow \infty} \mathbf{y}(t) = \lim_{t \rightarrow \infty} \mathbf{r}(t)$$

assuming a
single input
for simplicity

which makes $e_{ss} = 0$ (zero steady state error)

Full state feedback with reference input

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$$\lim_{t \rightarrow \infty} y(t) = \lim_{t \rightarrow \infty} r(t)$$

$$\lim_{s \rightarrow 0} sY(s) = \lim_{s \rightarrow 0} sR(s)$$

Final value theorem

$$\lim_{s \rightarrow 0} \frac{Y(s)}{R(s)} = 1$$



Closed loop transfer function at DC equals 1

Full state feedback with reference input

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Illustrative Example

Find the feedback gains \mathbf{K} of the following SS system such that the closed loop poles become $-2+2i$ and $-2-2i$, then find the closed loop TF at DC

$$\dot{\mathbf{x}}(t) = \begin{bmatrix} 0 & 1 \\ -1 & -1 \end{bmatrix} \mathbf{x}(t) + \begin{bmatrix} 0 \\ 1 \end{bmatrix} \mathbf{u}(t)$$
$$y(t) = \begin{bmatrix} 1 & 0 \end{bmatrix} \mathbf{x}(t)$$

Solution

From last lecture $\mathbf{K} = [7 \quad 3]$

$$\dot{\mathbf{x}}(t) = (\mathbf{A} - \mathbf{BK}) \mathbf{x}(t) + \mathbf{B}r(t)$$

$$\dot{\mathbf{x}}(t) = \begin{bmatrix} 0 & 1 \\ -8 & -4 \end{bmatrix} \mathbf{x}(t) + \begin{bmatrix} 0 \\ 1 \end{bmatrix} r(t)$$

Full state feedback with reference input

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Illustrative Example

Find the feedback gains \mathbf{K} of the following SS system such that the closed loop poles become $-2+2i$ and $-2-2i$, then find the closed loop TF at DC

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$$y(t) = \begin{bmatrix} 1 & 0 \end{bmatrix} \mathbf{x}(t)$$

Solution

From last lecture $\mathbf{K} = [7 \quad 3]$

$$\therefore TF = \frac{Y(s)}{R(s)} = \mathbf{C}(s\mathbf{I} - \mathbf{A}_{cl})^{-1} \mathbf{B} = \mathbf{C}(s\mathbf{I} - (\mathbf{A} - \mathbf{BK}))^{-1} \mathbf{B}$$

$$= \begin{bmatrix} 1 & 0 \end{bmatrix} \begin{bmatrix} s & -1 \\ 8 & s+4 \end{bmatrix}^{-1} \begin{bmatrix} 0 \\ 1 \end{bmatrix} = \frac{1}{s^2 + 4s + 8} \cdot \begin{bmatrix} 1 & 0 \end{bmatrix} \begin{bmatrix} s+4 & 1 \\ -8 & s \end{bmatrix} \begin{bmatrix} 0 \\ 1 \end{bmatrix} = \boxed{\frac{1}{s^2 + 4s + 8}}$$

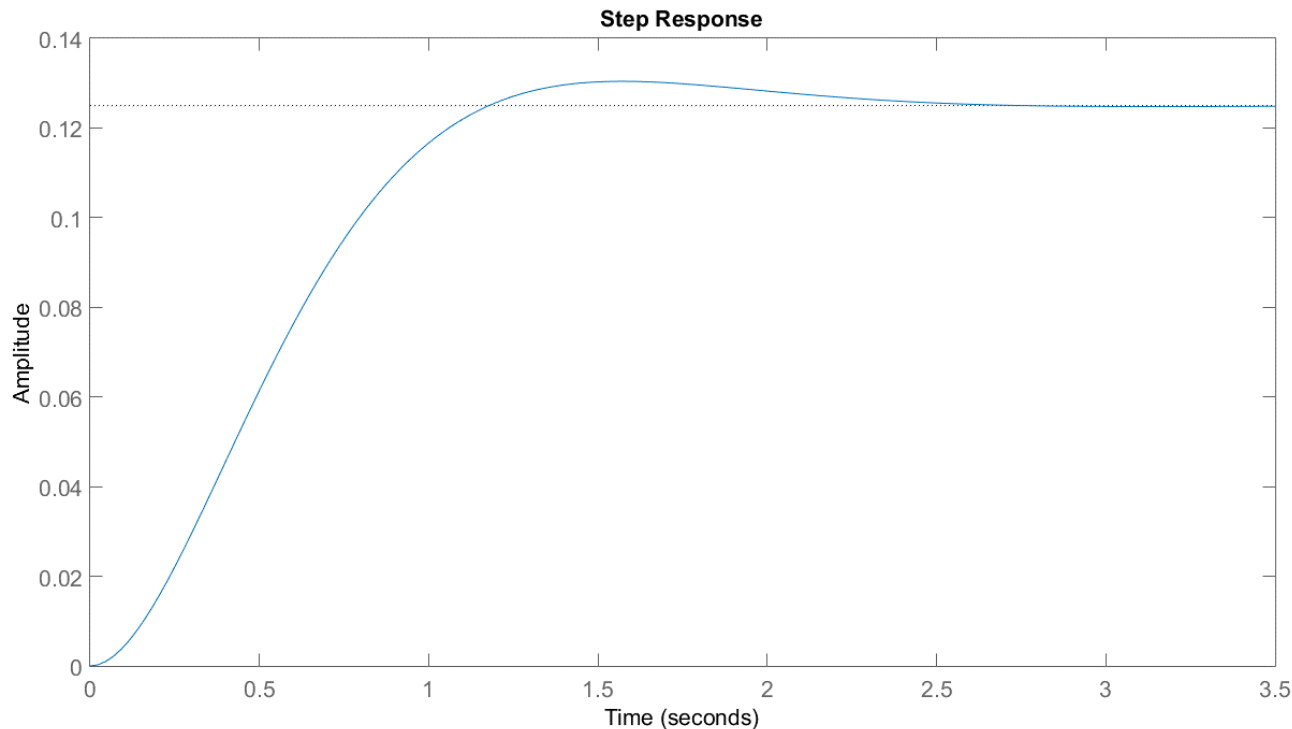
Full state feedback with reference input

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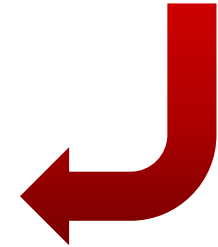
Illustrative
Example

$$\therefore \lim_{s \rightarrow 0} \frac{Y(s)}{R(s)} = \frac{1}{8} \neq 1$$

```
sys = ss(A-B*K,B,C,D)  
step(ss)
```



$y_{SS} = 0.125$
 $r_{SS} = 1$



How can we
overcome this
lack of
tracking?

Pre-scaling reference input

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Illustrative
Example

$$\dot{\mathbf{x}}(t) = \begin{bmatrix} 0 & 1 \\ -1 & -1 \end{bmatrix} \mathbf{x}(t) + \begin{bmatrix} 0 \\ 1 \end{bmatrix} \mathbf{u}(t)$$
$$y(t) = \begin{bmatrix} 1 & 0 \end{bmatrix} \mathbf{x}(t)$$

Solution

If we pre-scale the reference input r by a factor \bar{N} before entering closed loop system

$$\dot{\mathbf{x}}(t) = (\mathbf{A} - \mathbf{BK}) \mathbf{x}(t) + \mathbf{B}\bar{N} r(t)$$
$$\dot{\mathbf{x}}(t) = \begin{bmatrix} 0 & 1 \\ -8 & -4 \end{bmatrix} \mathbf{x}(t) + \begin{bmatrix} 0 \\ 1 \end{bmatrix} \bar{N} r(t)$$
$$y(t) = \begin{bmatrix} 1 & 0 \end{bmatrix} \mathbf{x}(t)$$

Pre-scaling reference input

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Illustrative
Example

$$\dot{\mathbf{x}}(t) = \begin{bmatrix} 0 & 1 \\ -1 & -1 \end{bmatrix} \mathbf{x}(t) + \begin{bmatrix} 0 \\ 1 \end{bmatrix} \mathbf{u}(t)$$
$$y(t) = \begin{bmatrix} 1 & 0 \end{bmatrix} \mathbf{x}(t)$$

Solution

If we pre-scale the reference input r by a factor \bar{N} before entering closed loop system

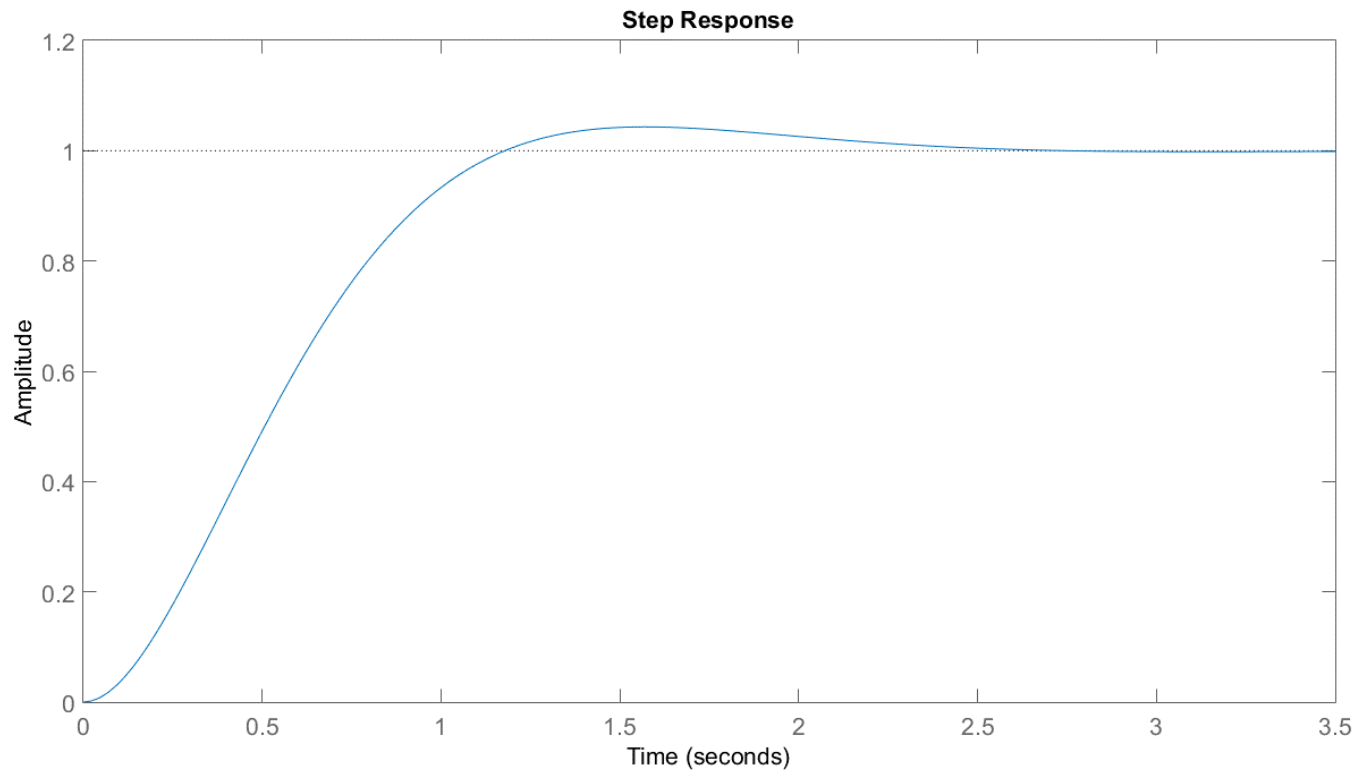
$$\begin{aligned} \therefore TF &= \frac{Y(s)}{R(s)} = \mathbf{C}(s\mathbf{I} - \mathbf{A}_{cl})^{-1} \bar{N} \mathbf{B} = \mathbf{C}(s\mathbf{I} - (\mathbf{A} - \mathbf{B}\mathbf{K}))^{-1} \bar{N} \mathbf{B} \\ &= \frac{\bar{N}}{s^2 + 4s + 8} \end{aligned}$$
$$\therefore \lim_{s \rightarrow 0} \frac{Y(s)}{R(s)} = \frac{\bar{N}}{8} \Rightarrow \boxed{\bar{N} = 8}$$

Pre-scaling reference input

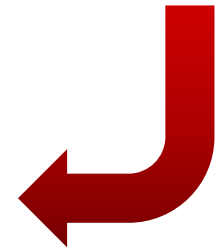
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Illustrative Example

```
sys = ss(A-B*K,Nbar*B,C,D)  
step(ss)
```



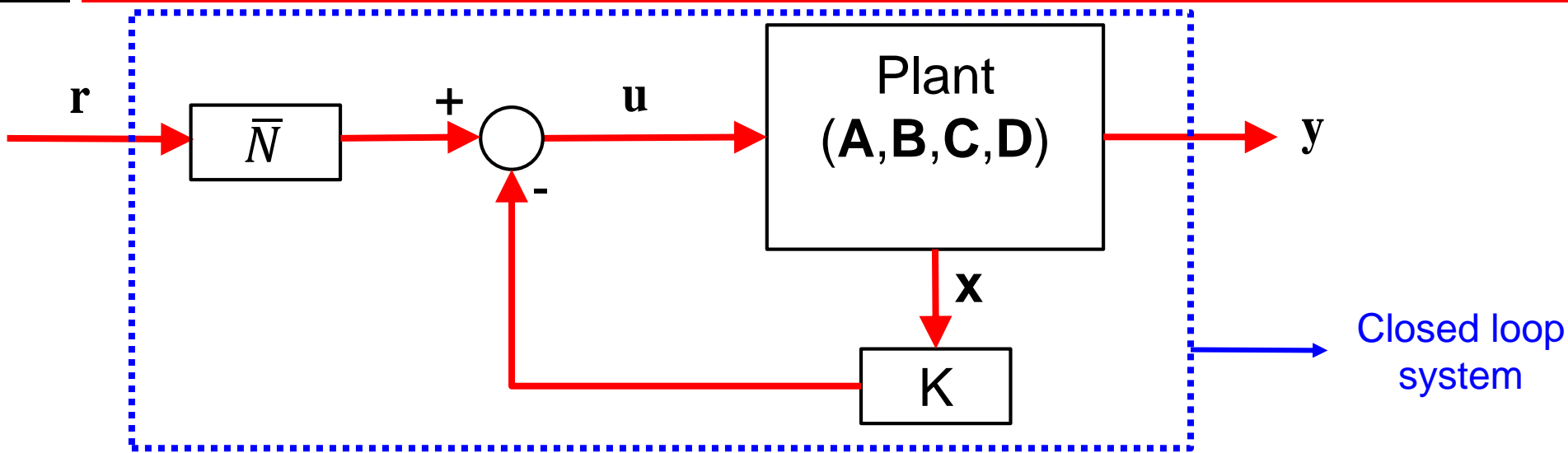
$$y_{ss} = 1$$
$$r_{ss} = 1$$



Problem solved with pre-scaling r

Full state feedback with pre-scaled reference (summary)

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Assuming full state feedback and pre-scaled reference input to achieve Steady state tracking

$$u(t) = \bar{N} r(t) - \mathbf{K} \mathbf{x}(t)$$

$$\begin{aligned} \dot{\mathbf{x}}(t) &= (\mathbf{A} - \mathbf{B}\mathbf{K}) \mathbf{x}(t) + \mathbf{B}\bar{N} r(t) \\ y(t) &= \mathbf{C}\mathbf{x}(t) \end{aligned}$$

$$\bar{N} = \frac{1}{\mathbf{C}(-\mathbf{A} + \mathbf{B}\mathbf{K})^{-1} \mathbf{B}}$$

MATLAB example 1 (2nd order system)

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$$G(s) = \frac{8}{(s - 5)(s - 10)}$$

Maximum overshoot = 5%

2% settling time = 4 sec

- Find the desired two poles
- Find K that achieves so (same as last lecture)
- Find Nbar that achieves steady state tracking
- Make sure closed loop system satisfy the specifications

MATLAB example 2 (3rd order system)

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$$G(s) = \frac{8}{(s-5)(s-10)(s-7)}$$

Maximum overshoot = 5%

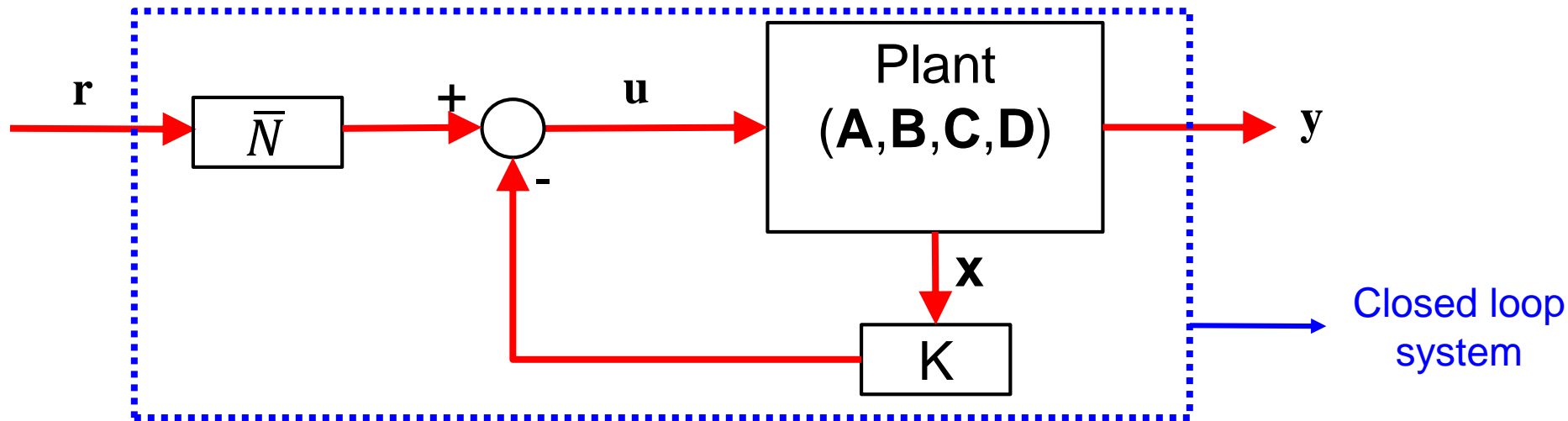
2% settling time = 4 sec

- Find the desired two poles
- Place the remaining pole far away
- Find K that achieves so (same as previous lecture)
- Find Nbar that achieves steady state tracking
- Make sure closed loop system satisfy the specifications

Intro to State estimators / observers

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- What we did so far



- Problem is that we have assumed **full state feedback** which means we have full access to the state variables of the system from which $u = \bar{N} r - \mathbf{K} \mathbf{x}$ is evaluated
- This is not true since in reality we only have access to the sensor outputs y and not the state variables \mathbf{x}
- Could try output feedback but will have less degrees of freedom compared to state feedback (cannot control all pole locations freely like what we did with \mathbf{K})

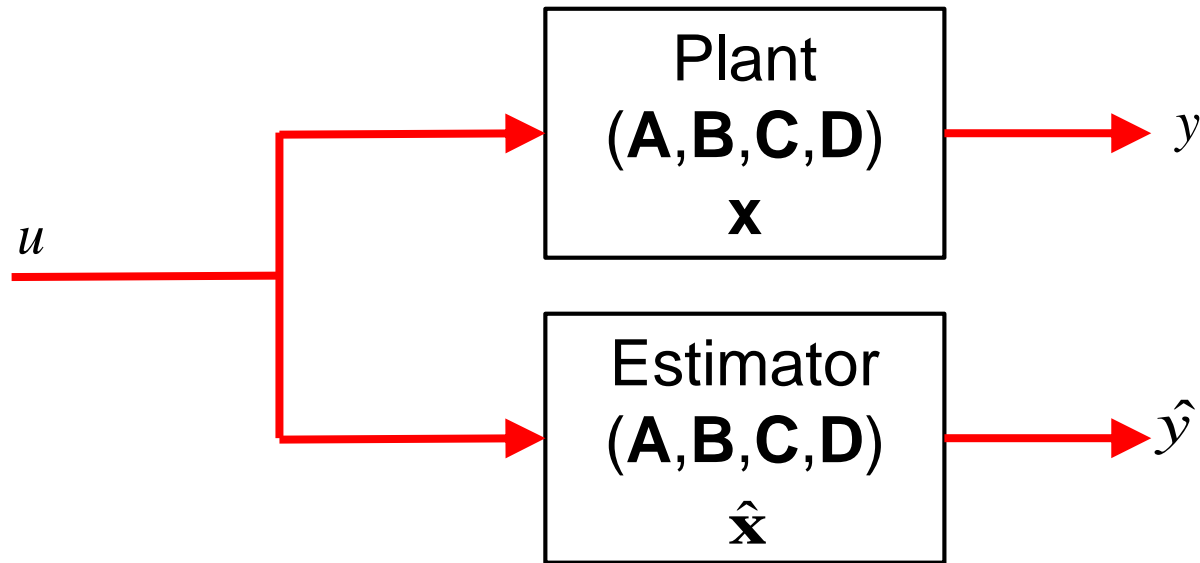
Intro to State estimators / observers

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- The solution to the lack of measurements of \mathbf{x} is to use **state estimator/observer**
- A state estimator/observer is a replica of the actual system or plant that tries to estimate the true state variables of the system from the actual measured output y and provides the estimated state vector $\hat{\mathbf{x}}$
- We can then combine the developed estimator together with state feedback control to have a realistic method of controlling the closed loop poles based on the feedback of estimated state variables $u = -\mathbf{K}\hat{\mathbf{x}}$ (more on this later but we will focus on estimator alone for the moment)
- Estimation strategies we have in hand
 - Open loop (bad strategy as we will see)
 - Closed loop

Open loop estimator

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- Assuming we know the input u and plant matrixes \mathbf{A} , \mathbf{B} , \mathbf{C} , and that $\mathbf{D} = 0$
- We can just simulate a replica of the actual plant on say a computer and obtain an estimate $\hat{\mathbf{x}}$ as follows

$$\dot{\mathbf{x}}(t) = \mathbf{A}\mathbf{x}(t) + \mathbf{B}u(t)$$

$$\dot{\hat{\mathbf{x}}}(t) = \mathbf{A}\hat{\mathbf{x}}(t) + \mathbf{B}u(t)$$

Open loop estimator

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$$\dot{\mathbf{x}}(t) = \mathbf{A}\mathbf{x}(t) + \mathbf{B}u(t) \quad \leftarrow \text{Dynamic eq. of actual plant}$$

$$\dot{\hat{\mathbf{x}}}(t) = \mathbf{A}\hat{\mathbf{x}}(t) + \mathbf{B}u(t) \quad \leftarrow \text{Dynamic eq. of simulated plant (estimator)}$$

$$\text{If } \mathbf{x}(0) = \hat{\mathbf{x}}(0), \quad \mathbf{x}(t) = \hat{\mathbf{x}}(t) \quad \forall t$$

- However we do not know $\mathbf{x}(0)$ so how well the above estimator works if the initial estimation error is not zero
- Define the estimation error $\mathbf{e}(t)$

$$\mathbf{e}(t) = \mathbf{x}(t) - \hat{\mathbf{x}}(t)$$

$$\frac{d}{dt} \{ \mathbf{x}(t) - \hat{\mathbf{x}}(t) \} = \mathbf{A} (\mathbf{x}(t) - \hat{\mathbf{x}}(t))$$

$$\dot{\mathbf{e}}(t) = \mathbf{A}\mathbf{e}(t)$$

$$\therefore \mathbf{e}(t) = e^{\mathbf{A}t} \mathbf{e}(0)$$

Open loop estimator

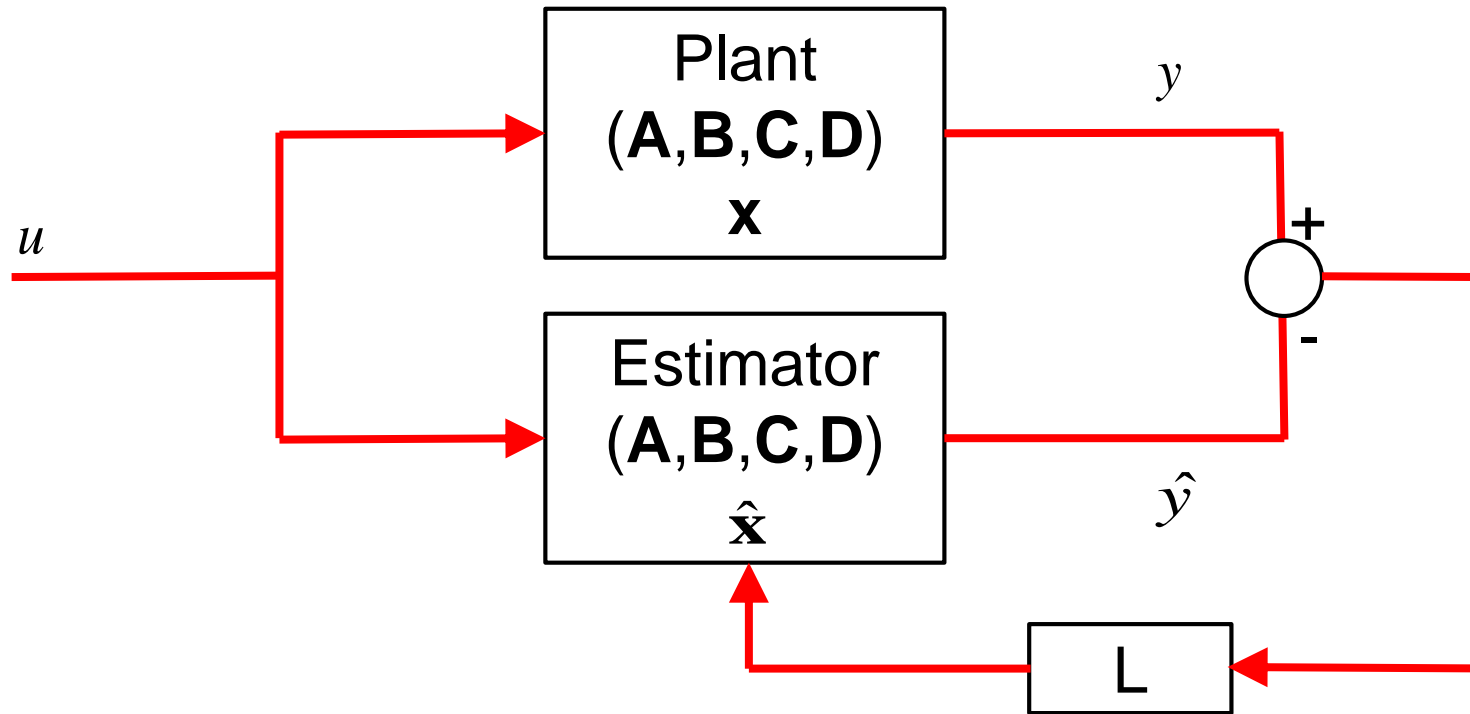
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$$\mathbf{e}(t) = e^{\mathbf{A}t} \mathbf{e}(0)$$

- Everything looks fine if initial error $\mathbf{e}(0) = 0$
- If $\mathbf{e}(0) \neq 0$, $\mathbf{e}(t)$ as $t \rightarrow \infty$ may decay to zero if the eigenvalues of \mathbf{A} have negative real part (if the original plant is stable)
- Since the estimation error is totally dependent on \mathbf{A} , this is not a good estimation strategy since we cannot control the dynamics of the estimation error at all
- We may make use of other available information in building a better state estimator (how? \rightarrow closed loop estimator)

Closed loop estimator

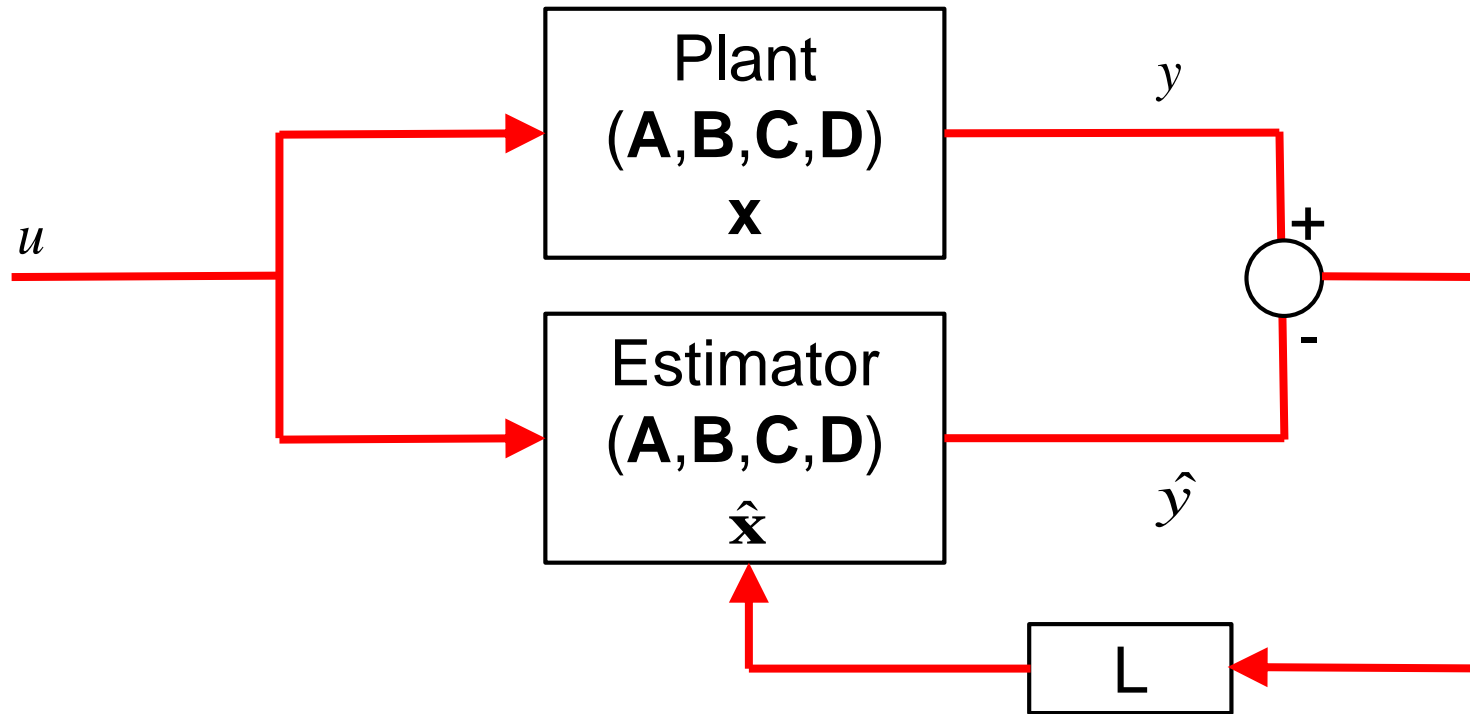
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- The idea is to feedback the error in the estimated output, i.e. its difference from the actual output of the system which can be observed
- L is a selectable gain matrix (similar to K) that will allow us to control the dynamics of the estimation error $e(t)$ as will be seen

Closed loop estimator

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$$\dot{\mathbf{x}}(t) = \mathbf{A}\mathbf{x}(t) + \mathbf{B}u(t)$$

Dynamic eq. of actual plant

$$\dot{\hat{\mathbf{x}}}(t) = \mathbf{A}\hat{\mathbf{x}}(t) + \mathbf{B}u(t) + \mathbf{L}(y(t) - \hat{y}(t))$$

Dynamic eq. of simulated plant (estimator)

Closed loop estimator

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$$\dot{\mathbf{x}}(t) = \mathbf{A}\mathbf{x}(t) + \mathbf{B}u(t)$$



Dynamic eq. of actual plant

$$\dot{\hat{\mathbf{x}}}(t) = \mathbf{A}\hat{\mathbf{x}}(t) + \mathbf{B}u(t) + \mathbf{L}(y(t) - \hat{y}(t))$$



Dynamic eq. of simulated plant (estimator)

$$y(t) = \mathbf{C}\mathbf{x}(t)$$



Output eq. of actual plant

$$\hat{y}(t) = \mathbf{C}\hat{\mathbf{x}}(t)$$



Output eq. of estimator

- Let's try to find the dynamics of $\mathbf{e}(t)$ with the added feedback to the estimator

$$\begin{aligned}\dot{\mathbf{e}}(t) &= \dot{\mathbf{x}}(t) - \dot{\hat{\mathbf{x}}}(t) = \mathbf{A}(\mathbf{x}(t) - \hat{\mathbf{x}}(t)) - \mathbf{L}(y(t) - \hat{y}(t)) \\ &= \mathbf{A}\mathbf{e}(t) - \mathbf{L}\mathbf{C}(\mathbf{x}(t) - \hat{\mathbf{x}}(t)) \\ &= (\mathbf{A} - \mathbf{L}\mathbf{C})\mathbf{e}(t)\end{aligned}$$

Closed loop estimator

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$$\dot{\mathbf{e}}(t) = (\mathbf{A} - \mathbf{L}\mathbf{C})\mathbf{e}(t)$$

$$\therefore \mathbf{e}(t) = e^{(\mathbf{A} - \mathbf{L}\mathbf{C})t} \mathbf{e}(0)$$

- It is obvious that by choosing a proper gain matrix \mathbf{L} , we can control the dynamics of the estimation error, i.e. make it go to zero fast such that the estimated state variables converge to the actual state variables fast enough
- This is all controlled by the eigenvalues of $\mathbf{A} - \mathbf{L}\mathbf{C}$

$$|s\mathbf{I} - (\mathbf{A} - \mathbf{L}\mathbf{C})| = \prod_{j=1}^n (s - s_j) = 0$$


Desired pole locations of state estimator
where? \rightarrow we will see later

Controller and Observer design (Dual problems)

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Controller design


$$\mathbf{K} \in \mathbf{R}^{1 \times n}$$

$$|s\mathbf{I} - (\mathbf{A} - \mathbf{BK})| = \prod_{j=1}^n (s - s_j) = 0$$


Desired pole locations closed loop sys

Observer/Estimator design

$$\mathbf{L} \in \mathbf{R}^{n \times 1}$$

$$|s\mathbf{I} - (\mathbf{A} - \mathbf{LC})| = \prod_{m=1}^n (s - s_m) = 0$$


Desired pole locations of state estimator

- \mathbf{K} and \mathbf{L} are chosen to achieve desired pole locations
- **Controller** and **Observer** design are called **dual problems**
- Just like before when the system had to be **controllable** to find \mathbf{K} , the system now has to be **observable** to find \mathbf{L}

Akermann's formula for Observer design

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- It gives a formal way to obtain \mathbf{L}
- Without proof

$$\mathbf{L} = \mathbf{O}_n^{-1} \Phi_{desired} (\mathbf{A}) [0 \quad 0 \quad \dots \quad 1]^T$$

Observability
matrix

Desired characteristic
equation of state
estimator/observer

- Clearly \mathbf{O}_n needs to be invertible, hence full rank, hence the system must be **observable** in order to be able to find \mathbf{L}

Observer design

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Example

For the system with the following matrices and initial state vector

$$\mathbf{A} = \begin{bmatrix} -1 & 1.5 \\ 1 & -2 \end{bmatrix}, \quad \mathbf{C} = [1 \quad 0],$$

- Test observability
- Find \mathbf{L} that makes poles of estimator/observer at -3 and -4

Observer design

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Solution

For the system with the following matrices and initial state vector

$$\mathbf{A} = \begin{bmatrix} -1 & 1.5 \\ 1 & -2 \end{bmatrix}, \quad \mathbf{C} = [1 \quad 0],$$

- Test observability

$$\mathbf{O}_n = \begin{bmatrix} \mathbf{C} \\ \mathbf{CA} \end{bmatrix} = \begin{bmatrix} 1 & 0 \\ -1 & 1.5 \end{bmatrix} \Rightarrow \text{rank} \{ \mathbf{O}_n \} = 2 \Rightarrow \text{observable}$$

Observer design

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Solution

For the system with the following matrices and initial state vector

$$\mathbf{A} = \begin{bmatrix} -1 & 1.5 \\ 1 & -2 \end{bmatrix}, \quad \mathbf{C} = [1 \quad 0],$$

- Find \mathbf{L} that makes poles of estimator/observer at -3 and -4

$$|s\mathbf{I} - (\mathbf{A} - \mathbf{L}\mathbf{C})| = \prod_{m=1}^n (s - s_m) = 0$$

$$\left| \begin{bmatrix} s & 0 \\ 0 & s \end{bmatrix} - \begin{bmatrix} -1 & 1.5 \\ 1 & -2 \end{bmatrix} + \begin{bmatrix} L_1 \\ L_2 \end{bmatrix} [1 \quad 0] \right| = (s + 3)(s + 4)$$

$$\left| \begin{bmatrix} s + 1 + L_1 & -1.5 \\ L_2 - 1 & s + 2 \end{bmatrix} \right| = (s + 3)(s + 4)$$

Observer design

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Solution

For the system with the following matrices and initial state vector

$$\mathbf{A} = \begin{bmatrix} -1 & 1.5 \\ 1 & -2 \end{bmatrix}, \quad \mathbf{C} = [1 \quad 0],$$

- Find \mathbf{L} that makes poles of estimator/observer at -3 and -4

$$\left| \begin{bmatrix} s + 1 + L_1 & -1.5 \\ L_2 - 1 & s + 2 \end{bmatrix} \right| = (s + 3)(s + 4)$$

$$s^2 + (L_1 + 3)s + (2L_1 + 1.5L_2 + 0.5) = s^2 + 7s + 12$$

$$L_1 = 4$$

$$L_2 = 2.333$$

MATLAB

Use \mathbf{A}^T and \mathbf{C}^T as your \mathbf{A} and \mathbf{B} in “place” function
 $\mathbf{L} = \text{place}(\mathbf{A}', \mathbf{C}', \text{desired poles})$